

Application of Differential Geometry to study Mechanical Systems

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Some remembers

Remember your high school courses :

A mechanical system of mass m submitted to a force f can be modeled by :

$$x(t) \in \mathbb{R}^n, m \frac{d^2x}{dt^2} = f(x, \frac{dx}{dt})$$

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Mechanical Energy : $E_m = K + U$

Some remembers

Mechanical energy theorem : If all forces are conservative, then on the motions of the system :

$$\frac{dE_m}{dt}(x(t), \dot{x}(t)) = 0$$

Very useful to find the motions of conservative systems !

Limits of this model

We could maybe find systems that don't belong to \mathbb{R}^n

Or having some forces hard to describe with our classical geometry.

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But something can save us : **Differential geometry !**

Requirements

We will need some mathematical tools for this :

- ▶ Differentiable manifolds
- ▶ Vector Fields
- ▶ Differentiable Maps

We still need :

- ▶ Tensors, Tensor Fields, Riemannian metrics

Tensors

Let V a vector space, $k \in \mathbb{N}$

A k -tensor T is a k -multilinear map : $V^k \rightarrow \mathbb{R}$

Example : The euclidean scalar product $(a, b) = x_a x_b + y_a y_b$ is a 2-tensor

Tensors on tangent vectors

Let M a manifold and (x^1, \dots, x^n) a coordinate chart, $p \in M$.

Let $v = \sum_{i=1}^n v^i \left(\frac{\partial}{\partial x^i}\right)_p \in T_p M$

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dx^i is a 1-tensor on $T_p M$, $dx^i(v) = v^i$

Each 1-tensor can be written as $\sum_{i=1}^n T_i dx^i$

Tensor product

Let T a k -tensor and S a p -tensor.

Tensor product :

$$(T \otimes S)(v_1, \dots, v_k, w_1, \dots, w_p) = T(v_1, \dots, v_k)S(w_1, \dots, w_p)$$

Is a $k+p$ tensor.

2-tensors on T_pM

Every 2-tensor on T_pM can be written as $T = \sum_{i,j=1}^n T_{ij} dx^i \otimes dx^j$

Thus for $v, w \in T_pM$, $T(v, w) = \sum_{i,j=1}^n T_{ij} v^i w^j$

Tensor Fields

A k -tensor field on TM gives $\forall p \in M$ a tensor $T(p)$

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For example, a 1-tensor field has this form :

$$T(p) = \sum_{i=1}^n T_i(p) dx^i$$

Differential of a function

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is a 1-tensor field on TM .

Riemannian metric

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Then g is a **Riemannian metric** on M

(M, g) : Riemannian Manifold.

Generalisation of Mechanical Systems

Let (M, g) a Riemannian Manifold and f a 1-tensor field :

$$\forall p \in M, f = \sum_{i=1}^n f_i(p) dx^i$$

. The triple (M, g, f) is called a Mechanical System.

Conservative Mechanical Systems

f conservative $\Leftrightarrow f = -dU, U \in C^\infty(M)$.

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Kinetic energy of $v \in TM$: $K(v) = \frac{1}{2}g(v, v)$.

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Let $c : I \subset \mathbb{R} \rightarrow M$ a smooth curve.

$$\forall t \in I, \dot{c} \in T_{c(t)}M$$

Kinetical energy of a curve : $K(\dot{c}(t)) = \frac{1}{2}g(\dot{c}, \dot{c})$

Fundamental theorem : Newton Equation

Let $(M, g, \sum_{i=1}^n f^i dx^i)$ a mechanical system, (x^1, \dots, x^n) local coordinates on M and $(x^1, \dots, x^n, v^1, \dots, v^n)$ local coordinates induced on TM . Then :

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$$\frac{d}{dt} \left(\frac{\partial K}{\partial v^i}(c(t), \dot{c}(t)) \right) - \frac{\partial K}{\partial x^i}(c(t), \dot{c}(t)) = f^i(c(t), \dot{c}(t))$$

$$\forall i \in [1, n]$$

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If the system is conservative ($f = -dU$), we have :

$$\frac{d}{dt} \left(\frac{\partial K}{\partial v^i}(c(t), \dot{c}(t)) \right) - \frac{\partial K}{\partial x^i}(c(t), \dot{c}(t)) = - \frac{\partial U}{\partial x^i}(c(t))$$

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Allows us to study the motion of loads of mechanical systems !

Examples

We are now going to look at examples of conservative systems, to study their motions.

Example 1 : Free falling particle

We let a point of mass m falling to the ground

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Mechanical system : $(\mathbb{R}^2, m(dx \otimes dx + dy \otimes dy), -d(mgy))$.

Riemannian metric : Euclidian scalar product

Potential energy : Classic gravitational potential energy $U = mgy$.

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We have $K(x, y, v^x, v^y) = \frac{1}{2}m((v^x)^2 + (v^y)^2)$

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$$(c(t), \dot{c}(t)) = (x(t), y(t), \dot{x}(t), \dot{y}(t))$$

We have

$$\frac{\partial K}{\partial x} = \frac{\partial K}{\partial y} = 0$$

$$\frac{\partial K}{\partial v^x} = mv^x$$

$$\frac{\partial K}{\partial v^y} = mv^y$$

Example 1 : Free falling particle

The newton equation gives :

On x:

$$\frac{d}{dt}\left(\frac{\partial K}{\partial v^x}(x, y, \dot{x}, \dot{y})\right) = \frac{d}{dt}(m\dot{x}) = m\ddot{x} = -\frac{\partial U}{\partial x} = 0$$

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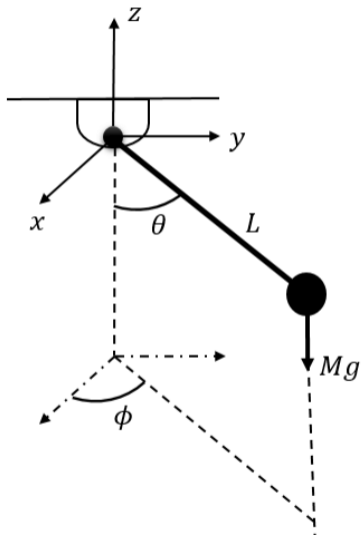
$$\frac{d}{dt}(m\dot{y}) = m\ddot{y} = -\frac{\partial U}{\partial y} = -mg$$

Two ODE's giving with initial conditions

$$x(t) = v_0^x t + x_0$$

$$y(t) = \frac{1}{2}gt^2 + v_0^y t + y_0$$

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The point is always far from l to the origin.

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Mechanical system : $(\mathbb{R}^3, m(dx \otimes dx + dy \otimes dy + dz \otimes dz), -d(mgz) + R)$.

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Two forces on p : His weight ($U = mgz$), and the reaction force of the rod.

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How to express the reaction force ?

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Theorem : If the rod makes no damping (amortecimento (pt), amortissement (fr), smorzamento (it)), by computing the motions of $(N, g, -dU)$

$$N = S_l^2 = \{p \in \mathbb{R}^3, x^2 + y^2 + z^2 = l^2\}$$

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How to do this ?

With spherical coordinates :

$$\Omega : (0, \pi) \times (0, 2\pi) \rightarrow \mathbb{R}^3$$

$$\Omega(\theta, \varphi) = (l \sin(\theta) \cos(\varphi), l \sin(\theta) \sin(\varphi), l \cos(\theta))$$

Example 2 : Spherical Pendulum

How to express the riemannian metric in spherical coordinates ?

Example 2 : Spherical Pendulum

In spherical coordinates :

$$g = g_{\theta\theta}d\theta \otimes d\theta + g_{\theta\varphi}d\theta \otimes d\varphi + g_{\varphi,\theta}d\varphi \otimes d\theta + g_{\varphi,\varphi}d\varphi \otimes d\varphi$$

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And we have :

$$g_{\theta\theta} = g\left(\frac{\partial\Omega}{\partial\theta}, \frac{\partial\Omega}{\partial\theta}\right)$$

$$g_{\theta\varphi} = g_{\varphi,\theta} = g\left(\frac{\partial\Omega}{\partial\theta}, \frac{\partial\Omega}{\partial\varphi}\right)$$

$$g_{\varphi\varphi} = g\left(\frac{\partial\Omega}{\partial\varphi}, \frac{\partial\Omega}{\partial\varphi}\right)$$

Example 2 : Spherical Pendulum

With

$$\frac{\partial \Omega}{\partial \theta} = (l \cos(\theta) \cos(\varphi), l \cos(\theta) \sin(\varphi), -l \sin(\theta))$$

And

$$\frac{\partial \Omega}{\partial \varphi} = (-l \sin(\theta) \sin(\varphi), l \sin(\theta) \cos(\varphi), 0)$$

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Example :

$$g_{\text{cartesian}} = m(dx \otimes dx + dy \otimes dy + dz \otimes dz)$$

$$g_{\theta\theta} = m(l^2 \cos^2(\theta) \cos^2(\varphi) + l^2 \cos^2(\theta) \sin^2(\varphi) + l^2 \sin^2(\theta))$$

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We obtain :

$$g_{\theta\theta} = ml^2$$
$$g_{\theta\varphi} = g_{\varphi,\theta} = 0$$
$$g_{\varphi\varphi} = ml^2 \sin^2(\theta)$$

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Newton Equations :

$$k(\theta, \varphi, v^\theta, v^\varphi) = \frac{ml^2}{2}((v^\theta)^2 + \sin^2(\theta)(v^\varphi)^2)$$

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$$\frac{d}{dt} \left(\frac{\partial K}{\partial v^\theta}(\theta, \varphi, \dot{\theta}, \dot{\varphi}) \right) = ml^2 \ddot{\theta}$$

$$\frac{d}{dt} \left(\frac{\partial K}{\partial v^\varphi}(\theta, \varphi, \dot{\theta}, \dot{\varphi}) \right) = ml^2 (\sin^2(\theta) \ddot{\varphi} + 2 \cos(\theta) \sin(\theta) \dot{\varphi} \dot{\theta})$$

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On θ

$$ml^2 (\ddot{\theta} - \sin(\theta) \cos(\theta) (\dot{\varphi})^2) = -\frac{\partial U}{\partial \theta} = mgl \sin(\theta)$$

On φ

$$ml^2 (\sin^2(\theta) \ddot{\varphi} + 2 \cos(\theta) \sin(\theta) \dot{\varphi} \dot{\theta}) = -\frac{\partial U}{\partial \varphi} = 0$$

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One solution : Numerical Methods

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How to solve these equations ?

One solution : Numerical Methods

- ▶ Euler Method
- ▶ Runge Kutta Methods

Gives good approximation of the solutions !

Runge Kutta example (Channel : Good Vibrations with Freeball)